

# IMPROVING AXIAL LOAD CARRYING CAPACITY OF COMPOSITE I-BEAM SECTION

Mr Jahirabbas K Shaikh<sup>1,2\*</sup>, Dr Anay Jain<sup>3</sup>, Dr Farook B Sayyad<sup>4</sup>

Research Scholar, Department of Mechanical Engineering, Faculty of Engineering, OPJS University, Churu, Rajasthan, India<sup>1</sup>

Lecturer, Department of Mechanical Engineering, Faculty of Technology, Debre Tabor University, Debre Tabor, Ethiopia<sup>2</sup>

Professor, Department of Mechanical Engineering, Faculty of Engineering, OPJS University, Churu, Rajasthan, India<sup>3</sup>

Professor, Department of Mechanical Engineering, Dr DYP School of Engineering, Lohegaon, Pune, Maharashtra, India<sup>4</sup>

**Abstract:** *Rusting of Polyurethane (PU) coated steel sample which is used in most of the chemical industries has become a problem requiring necessary attention and corrective prevention method. Composite materials are looked at as the alternatives to address this issue. As these composites are not attacked by the chemicals in the industries, acids in present problem. However, composites have complex manufacturing processes and need high cost. The glass fibre plates, composites, cut in to the necessary lengths and dimensions. Then joined at Web Flange Joint (WFJ) with the help of adhesives to reduce the process complexity and cost. It is observed that Huntsman Araldite gives better results as adhesive to be used at WFJ. This is validated using Finite Element Analysis (FEA) with ANSYS tool. The Huntsman Araldite failure strength is much better at the WFJ area. In addition to this, cleats are used with 4 mm, 12 mm for increasing the WFJ area and improve the load carrying capacity of the composite I- Beam. The results of experiments have clearly shown that the glass fibre immersed in the acidic bath solution for certain period has no reaction as such. Hence shown much improved resistance to acidic attack and no rusting as such. Appreciable correlation between experimental and analytical results indicated improved performance of Composite Beam in given chemical (acidic) environment. It is recommended to use the composite I – Beams in acidic environments in place of steel beams to have improved rusting behaviour and long life of the beams.*

**Keywords:** *Acidic environment, Composite I-Beam, FEA with ANSYS, Glass fiber composite, Huntsman Araldite Adhesive.*

## I. INTRODUCTION

The I-

beam is made by two different section horizontal element is called as flange and the vertical element is called as web. I – beam is known by variety of commercial names in the application industries, with an I shape cross section. The flanges resist the bending moment and the web resists shear forces experienced by the beam. As suggested by the beam theories, the I beams are very efficient in case of shear forces and bending moments. However, the cross section of these beams is inefficient in case of torsional stresses or twisting moments. Hence hollow sections are suggested in such loading conditions for these beams to get improved strengths.

There are two different processes used to make the I-beam sections. 1. Plate girder made by welding plates. 2. Rolled I-beam, made by extrusion, cold rolling, hot rolling. The major application areas of these beams are construction work in civil engineering work, mechanical industry, chemical, and automobile. In the present proposed work, I-beam which is used in chemical factory for stirrer mounting mechanism, is subjected to study. Fig. 1 shows that the geometry of I-beam is taken from ISMB 80 standard for using the sample of stirrer mounting mechanism. Height of the I-beam is 80 mm, flange width 50 mm and Web and Flange thickness is 5 mm.

Construction work, chemical industry, aircraft industry, automobile bodies are the major application areas where I beam are the structural members. In larger sections this I-beam is made up of steel with PU coating. Web-flange junction of that beam is joining by welding or nut bolt joint. This I beam is completely immersed into the acid, named benzoic acid. After certain duration, the PU coating shows rusting, as a result of reaction of the acidic environment with the base steel. This affected beam loses its utility and becomes a waste, cannot be used for any other application in future. Hence, as an alternative to this, the I-Beam with composite beam, made up of epoxy glass fiber material. However, the failure is mostly observed at the Web Flange Joint.

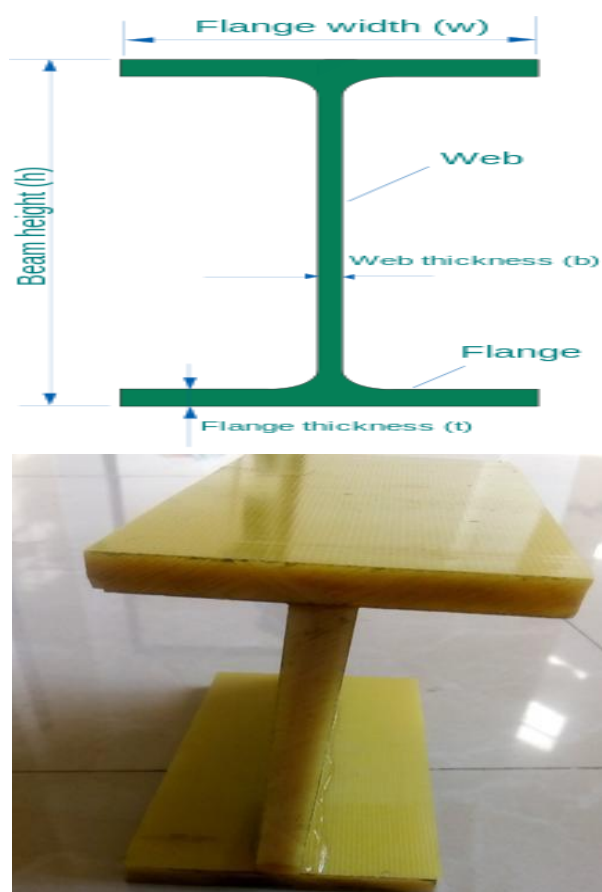


Fig. 1. Geometry of I-Beam model

Adhesives are used to prevent this problem occurrence. There are two types of adhesives used: 1<sup>st</sup> is Araldite 2015 and 2<sup>nd</sup> is Huntsman Araldite. A choice is made between these two adhesives. The stirrer and its motor are mounted on

that beam and stirrer is rotated at 5-6 rpm. The beam observed to be failing after some time at WFJ, the complete structure is immersed inside the acidic solution. The area at the joint is less and causing more stress concentration, which leads to failure. Hence it is suggested to use cleats of 4 mm, 12 mm, 16 mm at the joint to increase the area and cause more strengthening.

## II. METHODOLOGY

The main object of this research work is to obtain the structural design of I-beam, using reinforced epoxy glass fiber composite materials. This will address the problem of chemical reaction with steel. This will eliminate the WFJ failure with application of the adhesive. The mathematical calculations are carried out, using the empirical relations and conventional design formulae. A model is constructed to simulate the geometry and material properties of composite I Beam. This model is adapted as Finite element model in ANSYS workbench. Fine mesh with higher order element is generated to achieve the mesh convergence. This epoxy glass fiber composite I beam model is analysed using ANSYS workbench.

## III. PROBLEM STATEMENT

It is observed in normal practice, that, the I beam is completely immersed into the acid. This acid is benzoic acid and after some time the PU coating is showing rust on its surface. This is effect of reaction of the acid with I beam material i.e. steel.

As we know in I beam structure, shear resistance is provided by the web while the bending moments are resisted by the flanges. Most of the times failure occurs at joint section of web and flange, known as web – flange joint (WFJ). Hence, this WFJ is an essential part of analysis in identifying the failure cause and its remedial action.

Hence this paper attempts to find out the design of structural I-beam, using reinforced epoxy glass fiber composite materials. As epoxy glass fiber is resistant to the acidic attack. This will address the issue of chemical reaction between the acid and the steel. To calculate the stress and force of composite I-beam at web-flange junction using FE analysis. The failure strength of WFJ is improved by using Huntsman araldite.

## IV. NUMERICAL FORMULATION

The type of the composite I-beam in simply supported type. I-beam is taken from ISMB 80 standard to be used as sample of stirrer mounting mechanism. Height of the I-beam is 80 mm, flange width 50 mm and Web and Flange thickness 5 mm. These dimensions are used for calculating the cross sectional area and the stress generated at Web – Flange - Joint.

**Table-1: Dimension of composite I-beam**

Beam Type	Height of beam (mm)	Width of flange (mm)	Thickness of Web (mm)	Flange Thickness (mm)
ISMB80	80	50	5	5

As shown in above Table 1 the dimension of composite I-beam. This dimension of I-Beam is selected from the Indian standard medium weight beam chart. This beam is ISMB 80 & this dimension is standard.

1) **Area of flange and web** =  $b \cdot d$

Where,

b = breadth

d = depth

2) **Total area of beam** =  $A_1 + A_2 + A_3$  Where,

$A_1$  &  $A_3$  = area of flange  $A_2$  = area of web

3) **Total area of beam (A)** =  $850 \text{ mm}^2$

**Moment of inertia (I)** =  $I_1 + I_2 + I_3$

$I_1 = \frac{bd^3}{12} + A_1(Y_{\max} - Y_1)$  Where,

$Y_{\max}$  = Distance from bottom to N.A.  $Y_1$  = distance from flange bottom to

flange N.A. **Moment of inertia (I)** =  $7.646 \text{ E}6 \text{ mm}^4$

Stress =  $P/A$

Where, P = force

A = Area of beam

## V. FINITE ELEMENT ANALYSIS (FEA)

The finite element model of composite I-beam is analysed using ANSYS. The finite element analysis techniques applied in this work were explained in a typical FEA. Consequently 80%-90% of the time is spent to find the best parameter settings. Finally, a three-dimensional, compatible FEA model was created in ANSYS workbench, and the results were compared with the physical test results. The geometry and material properties were same during either analyses to verify the accuracy of the FEA model.

### Material Properties

**Table-2: Properties of glass Fiber-Reinforcement Materials**

Fiber Property	Glass Fibers
Diameter ( $\mu\text{m}$ )	8-14
Density ( $\text{kg/m}^3$ )	2560
Modulus of Elasticity (GPa)	76
Tensile strength (GPa)	1.4-2.5
Elongation at fracture (%)	1.8-3.3

As shown in above Table 2 shows that the material properties of material of glass fiber which is used as reinforcement material.

**Table-3: Properties of Epoxy matrix Materials**

MatrixProperty	Epoxy
Density(kg/m <sup>3</sup> )	1100-1400
ModulusofElasticity(GPa)	03-Jun
TensileStrength(GPa)	0.035-0.10
CompressiveStrength(GPa)	0.1-0.2
ElongationatFracture(%)	01-Jun

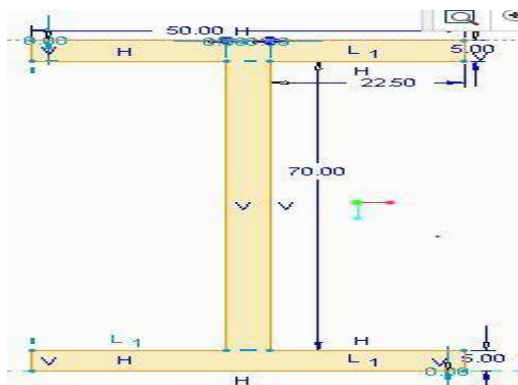
As shown in above Table 3 the material properties of Epoxy and which is used as matrix material for producing the glassfiber epoxyresinmaterial.

**FEModelinformation:**

The geometric model is built to simulate the actual stirrer mounting mechanism, as part under study. The boundary conditions also resemble to the actual loading conditions. Hence, a non-linear FEA has been carried out to to achieve the high level of accuracy. Multi-dimensional stress - strain curve, frictional contact between the mating parts and geometric non-linearity has been defined in FE model. ANSYS workbench is used to carry out this non-linear analysis. Firstly, finite element analysis has been carried out on current design (baseline model). The fine mesh is used so as to increase the accuracy of results.

**2D Modeling of I-Beam:**

CAD software like CREO which is used for generate 2D model of the project .We can use CREO for generate the I-beam model and which is converted into IGES format. Fig2 show that the Creo model of the I-beam.



**Fig.2.Creo design of I beam Loads and Boundary conditions**

This I-beam is used in acid chemical factory for stirrer mounting mechanism in these flanges are connected at the tank & the load bearing is mounted at the web in this one end of beam is fixed and load is applied at an other end this load is the tensile. This load is the motor and stirrer weight which is mounted at the I-beam. Load and boundary condition is shown in the Fig.3.

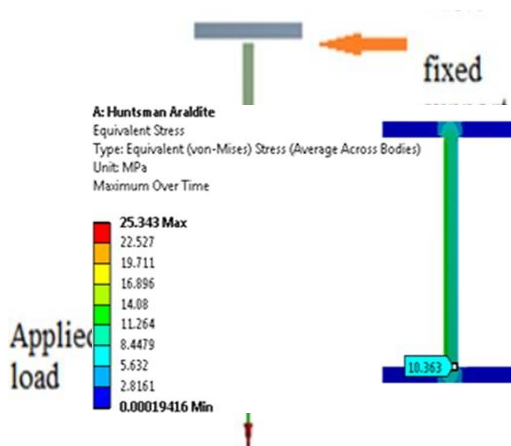


Fig. 3. Loads and Boundary conditions Analysis result of I-beam model:

The conventional model is developed into the CREO software and this model converted into IGE S format which is imported into the ANSYS software for the analysis purpose. There are two adhesives used for joining the WFJ 1st one is Araldite 2015 and 2nd one is Huntsman Araldite and then find out which is better for joining WFJ of composite I-beam.

I-Beam with Araldite 2015 Adhesive:

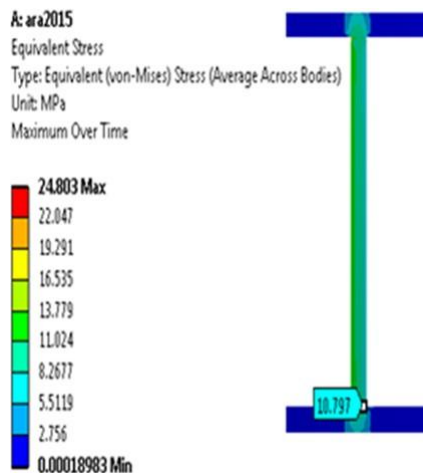


Fig. 4. Von Mises stress with Araldite 2015

The above Fig. 4 shows the Von Mises stress plot. The max Von Mises stress occurred on WFJ is 10.79 MPa and load carrying capacity is 14945 N.

I-Beam with Huntsman Araldite Adhesive

Following Fig. 5 shows the Von Mises stress plot and max load carrying capacity plot. The max Von Mises stress occurred on WFJ is 10.363 MPa and load carrying capacity is 15269 N.

Fig.5 Vonmises stress withHundsmanaraldite Analysisresultsof nonlinearmodel

Now in this step, FE model is solved by considering the two different adhesives with composite I-beam. Table.4 shows that the maximum stress concentration at WFJ of the I-beam with Araldite 2015 and Hundsmanaraldite.

Table-4: Stress concentration at WFJ of I-Beam

Sr. No	I-Beam with adhesive	Stress (MPa)
1	Araldite 2015	10.79
2	Hundsmanaraldite	10.36

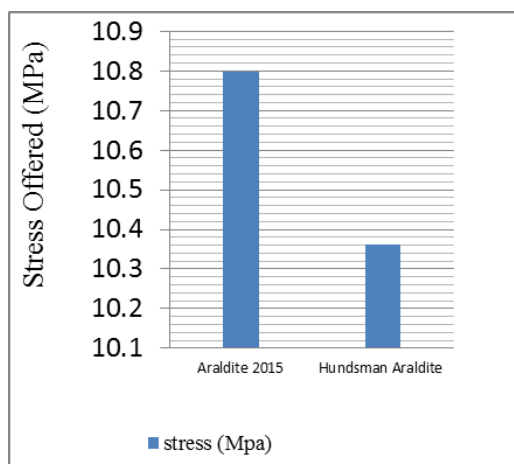


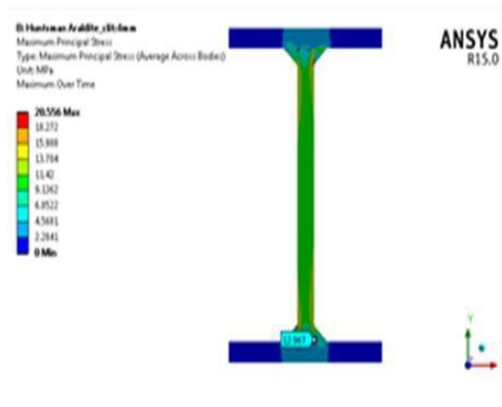
Fig.6. Stress concentration of I-beam at WFJ with adhesive

As shown in above Fig.6 the graphical representation of stress concentration of I-beam at WFJ with adhesive.

From above Table 4 and Fig.6 shows that the Hundsmanaraldite is a better adhesive material as compared to the Araldite 2015 because its stress concentration at WFJ is less and load carrying capacity is more. Stress concentration at WFJ of the Hundsman araldite is 10.36 MPa and Araldite 2015 is 10.79 MPa and force capacity of Hundsman araldite is 15269 N and Araldite 2015 is 14945.

In this I-beam structure there is less area of joining at WFJ then the beam is failure at the junction to avoid this problem we can use the extra cleat which dimension is 4mm and 16mm which is available in the market for increasing the area of joining which increases the load carrying capacity of the beam.

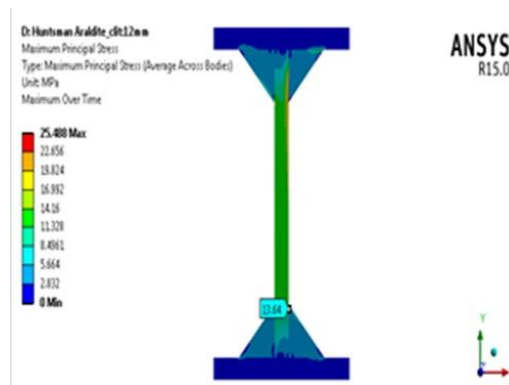
**I-BeamwithHundsmanAralditeand4mmCleat**



**Fig.7StressofI-beamHundsmanaralditewith 4mmcleat**

As shown in above Fig.7the Von Mises stress plot. The max Von Mises stress occurred on WFJ is 12.96 MPa and load carrying capacity is 17018N.

**I-BeamHundsmanAralditeAdhesivewith12mmCleat**



**Fig.8StressI-beamHundsmanaralditewith12mmcleat**

As shown in above Fig.8the Von Mises stress plot. The max Von Mises stress occurred on WFJ is 13.64MPa and load carrying capacity is 18602N.

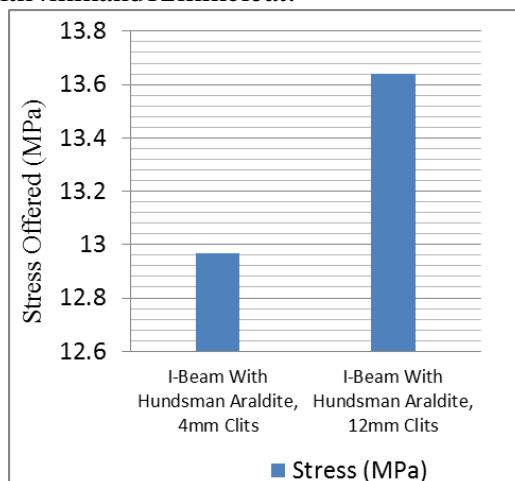
**Table-5: Stress carrying capacity of composite I-beam with Huntsman araldite**

Sr.No	I-Beam specimen with different size cleat	Stress (MPa)
1	I-Beam With Huntsman Araldite, 4mm Cleat	12.96



2	I-Beam WithHundsmanAraldite,12mmCleat	13.64
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As shown in above Table.5 the maximum stress carrying capacity of composite I-beam huntsman araldite with 4mm and 12mm cleat.



**Fig.9. Stress sustained by I-Beam Hundsman araldite with 4mm and 12 mm cleat**

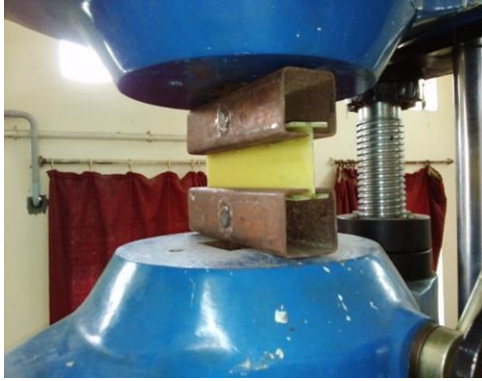
As shown in above Fig. 9 the graphical representation of stress carrying capacity of I-beam with 4mm and

12mm cleat. From the Table. 5 and Fig. 9 shows that when we increase the size of cleat then the stress and load carrying capacity of beam increases.

## I. PHYSICAL TESTING:

As shown in Fig.10 the Universal Testing Machines (UTM): In this project we choose UTM for physical testing of the specimen. UTM is used for taking four type of material test and my project test is tensile test. Universal Testing Machines are named because they can perform many different varieties of test on the material. Another name UTM is material testing machine and machine can be used to test tensile and compressive properties of materials. They can perform the various operations on same machines such as tension, compression, bending etc to examine mechanical properties of the material this type of machine is called as UTM. There are two types of machines available in the market like single column and two column in our project using two column machine. Digital display and extensometers are used for measuring the force and deformation which can also be presented in graphical form in computer-operated machine. These machines are widely used in industry and all material testing laboratory.





**Fig.10. Experimental set up for tensile test of I-Beam UTM**      **Fig.11 Manufactured sample of I-Beam**

As shown in above Fig.11 the actual manufactured specimen of I-beam and which is used for the testing.

#### **1st Sample I-beam with adhesive araldite 2015**

As shown in below Fig.12 and Fig 13 the composite I-beam with araldite 2015 before and after testing. Test is a tensile test the load is applied to hydraulic pressure

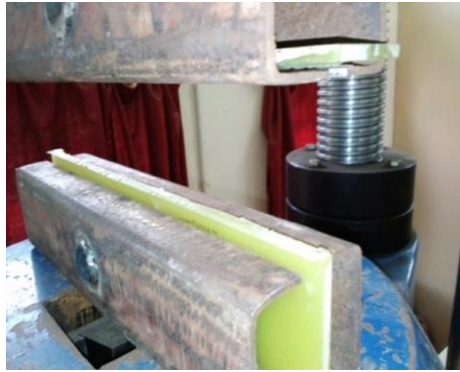


**Fig.12. I-beam with adhesive araldite 2015 before test**



**Fig.13. I-beam with adhesive araldite 2015 after test 2<sup>nd</sup> Sample I-beam with adhesive Huntsman araldite**

As shown in below Fig.14 and Fig.15 the composite I-beam with Huntsman araldite before and after testing.



**Fig 14.I-beam with adhesive Huntsman araldite before test**

**Table-6:Experimental stress of I-beam**

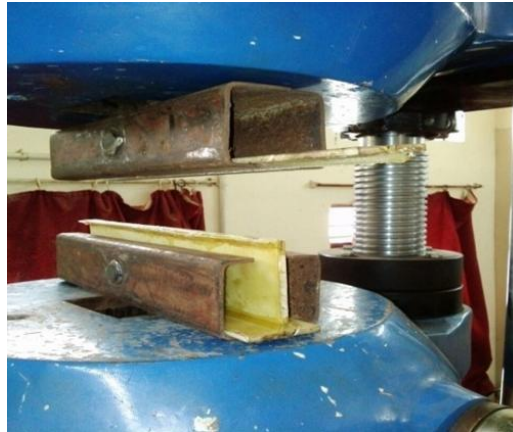
Sr. No	I-Beam with adhesive	Stress (MPa)
1	Araldite 2015	9.54
2	Hudson Araldite	9.83

As shown in above Table 6 the stress generated of manufactured I-beam with araldite 2015 and Huntsman araldite.



**Tensile Testing of 3rd I-Beam specimen with 4mm cleat**

**Fig.16.I-beam with 4mm cleat before Test**



**Fig17.I-beamwith4mmcleatAfterTest**

As shown in above Fig.16 and Fig.17 the composite I-beamHundsmanaralditewith4mmcleatbeforeandaftertesting.

**Tensile testing of 4th I-Beam specimen with 12mm cleat**



**Fig.18.CompositeI-Beamwith12mmcleatbeforeTest**



**Fig.19. Composite I-Beam with 12 mm cleat After Test**

As shown in above Fig. 18 and Fig. 19 the composite I-beam Huntsman Araldite with 12mm cleat before and after testing.

**Table-7: Max stress of I-beam with cleat**

Sr. No	I-Beam specimen with different size cleat	Stress (MPa)
1	I-Beam with hundsman Araldite, 4mm cleat	12.06
2	I-Beam with hundsman Araldite, 12mm cleat	12.46

As shown in above Table 7 the maximum stress sustained by I-beam Huntsman Araldite with 4mm and 12mm cleat.

### I. RESULT AND DISCUSSION

In this study an attempt is made to calculate the stress concentration at WFJ of joint. Then numerical

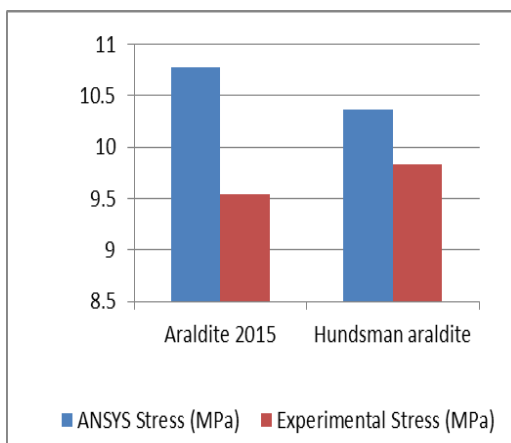
formulation has been established with the help of empirical formulae and analytically stress is calculated. The stress concentration calculated by analytical method is 10.36 MPa.

**Table.8: Maximum Stress concentration of I-Beam at WFJ**

Sr. No	I-Beam Specimens with Adhesive	ANSYS stress (MPa)	Analytical stress (MPa)
1	I-beam with araldite 2015	10.78	9.54
2	I-beam with Hundsman Araldite	10.36	9.83

Table 8 shows that the Ansys and Experimental stress sustained by the I-beam with adhesive araldite 2015 and Huntsman Araldite.

As shown in Fig. 20 the graphical representation of I-beam Ansys and Experimental stress value of I-Beam with adhesive araldite 2015 and Huntsman Araldite



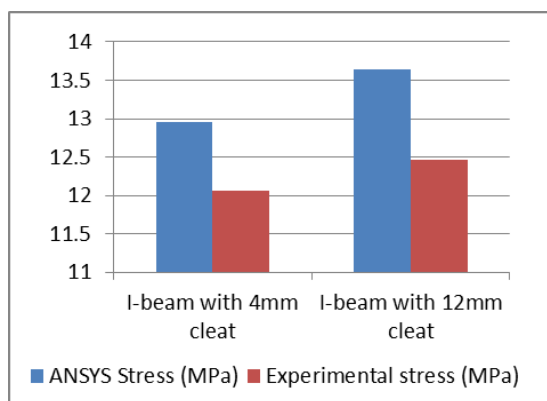
**Fig.20 Maximum Stress concentration of I-Beam at WFJ**

From Table 8 and Fig 20 shows that the Hundsman araldite is better adhesive for joining the WFJ because there is less stress concentration as compared to araldite 2015.

In next step to increase the stress and load carrying capacity of composite I-beam Hundsman araldite with the help of cleat. Table 9 shows that the Ansys and Experimental stress sustained by I-beam with the different size of cleat. Size of cleat is 4mm and 12mm.

**Table-9: Maximum Stress Sustained By I-Beam with cleat**

Sr.No	I-Beam Hundsman araldite with cleats	ANSYS Stress (MPa)	Analytical Stress (MPa)
1	I-beam with 4mm cleat	12.96	12.067
2	I-beam with 12mm cleat	13.64	12.46



**Fig.21 Maximum Stress Sustained By I-Beam with cleats**

Fig.21 shows that the graphical representation of Ansys and Experimental stress sustained

by I-beam. Table 9 shows that when we increase the size of cleat then the stress and load carrying capacity of beam.

The physical testing is carried out to verify the results obtained from finite element analysis. From physical testing the observed stress at WFJ of I-beam with Hunsman Araldite is 9.83 MPa and with cleat 4mm is 12.06 and 12mm is 12.46 MPa.

All the results obtained in this study are tabulated in following Table

**Table-10: Result summary**

I-beam with	Stress MPa	
	FEA	Exp
Araldite 2015	10.78	9.54
Hunsman Araldite	10.363	9.83
Hunsman Araldite with 4mm	12.96	12.06
Hunsman Araldite with 12mm	13.64	12.46

Physical testing results of baseline model it is observed that there is only 1.43% difference with FEA results. For the model confirmation testing is to be carried out FE analysis results and physical testing results.

## II. CONCLUSION

The present work has successfully demonstrated the application of Design of experiment. The final conclusions arrived, at the end of this work are as follows:

Composite I-

Beam will be best option in chemical factory because there is no reaction with acid. Also, its main advantage is low weight as compared to steel.

The composite I-beam with Hunsman Araldite is better than I-beam with Araldite 2015 because there is less stress concentration at WFJ and load carrying capacity is more.

The stress and load carrying capacity of composite I-beam with Hunsman Araldite can be increased by attaching different size cleats to web-flange junction.

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## AUTHORS PROFILE

**Jahirabbas K Shaikh**, he is presently Ph D Research Scholar in Mechanical Engineering Department, Faculty of Engineering, OPJS University, Churu, Rajasthan, India. He is also working as Lecturer at DTU, Ethiopia.

**Dr Anay Jain**, he is professor in Mechanical Engineering Department, Faculty of Engineering, OPJS University, Churu, Rajasthan, India. He has several research publications to his credit.

**Dr Farook B Sayyad**, he is Professor, Department of Mechanical Engineering, Dr DYP School of Engineering, Lohegaon, Pune, Maharashtra, India. He has several research publications to his credit.